

**Wieland Thermal Solutions<sup>®</sup>**  
PROVIDING EFFICIENCY



**AHR New Product &  
Technology Theater**

**Optimization of Copper  
Tubes for ACR Fin Coil  
Heat Exchangers**

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Industrial Tubes

Dallas, TX  
01/29/2013

# Agenda

- Ø Introduction of Wieland
- Ø Inner-grooved copper tubes for fin coil heat exchangers:
  - § Why using copper?
  - § Why using inner-grooved tubes?
- Ø Correlation of tube unit weight and inside surface enhancement
- Ø Potential of high performance inner-grooved tubes
- Ø Reduction of tube unit weight
  - § Light weight tube design
  - § Thinner bottom wall
- Ø Miniaturization: Effect of reducing tube outside diameter
- Ø Development of inner-grooved tubes on customer demand

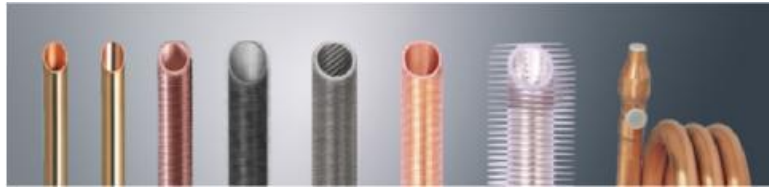


# Extruded and Drawn Products Division



## About Wieland

Manufacturer of semi-finished products



Headquartered in Germany



2 manufacturing sites in the US (NC and IL)



Founded in 1820

> 30 locations worldwide



Specialized in copper and copper alloys



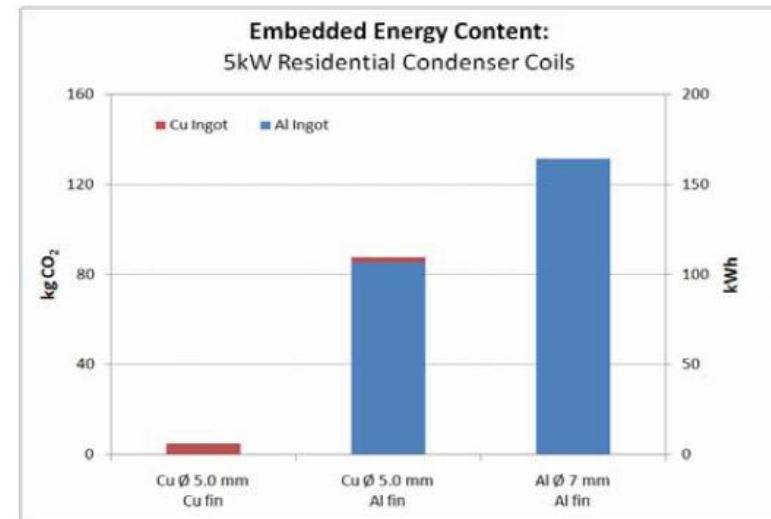
## Why using copper tubes for fin coil heat exchangers?

### Ø Corrosion and durability –

Copper coils feature superior and **reliable long-term performance**. Aluminum fin and tube combinations must be carefully matched and tested to provide sufficient corrosion protection and to maintain performance level long-term

### Ø Environmental impact –

Carbon footprint of copper is extremely low. Energy consumption to produce aluminum is extremely high.



Source: IBIS

Ø **Optimized tube technology** can be implemented into existing heat exchanger facilities immediately and **with only slight investments** into new equipment

Ø **Installation and assembly** (brazing) is very easy and safe

Ø **Ease of service and reliable repair** opportunity in the field

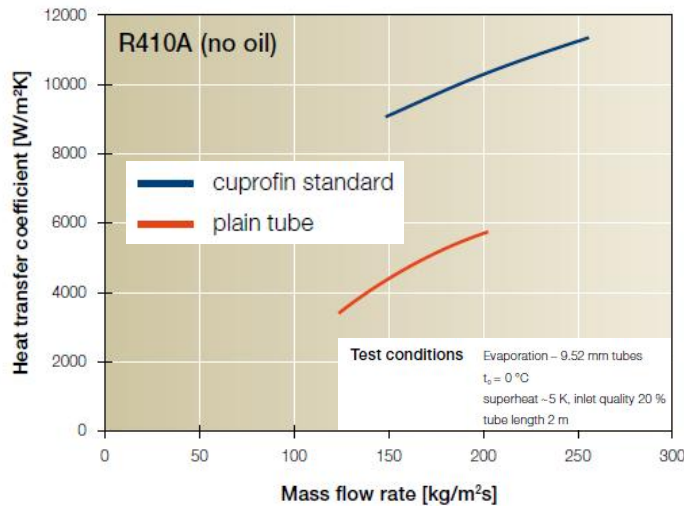
Ø **High flexibility** to modify heat exchanger size / capacity very easily and quickly

slide 4



## Why using inner-grooved copper tubes?

With inner-grooved tubes (IGT), the inside heat transfer performance can be increased considerably.

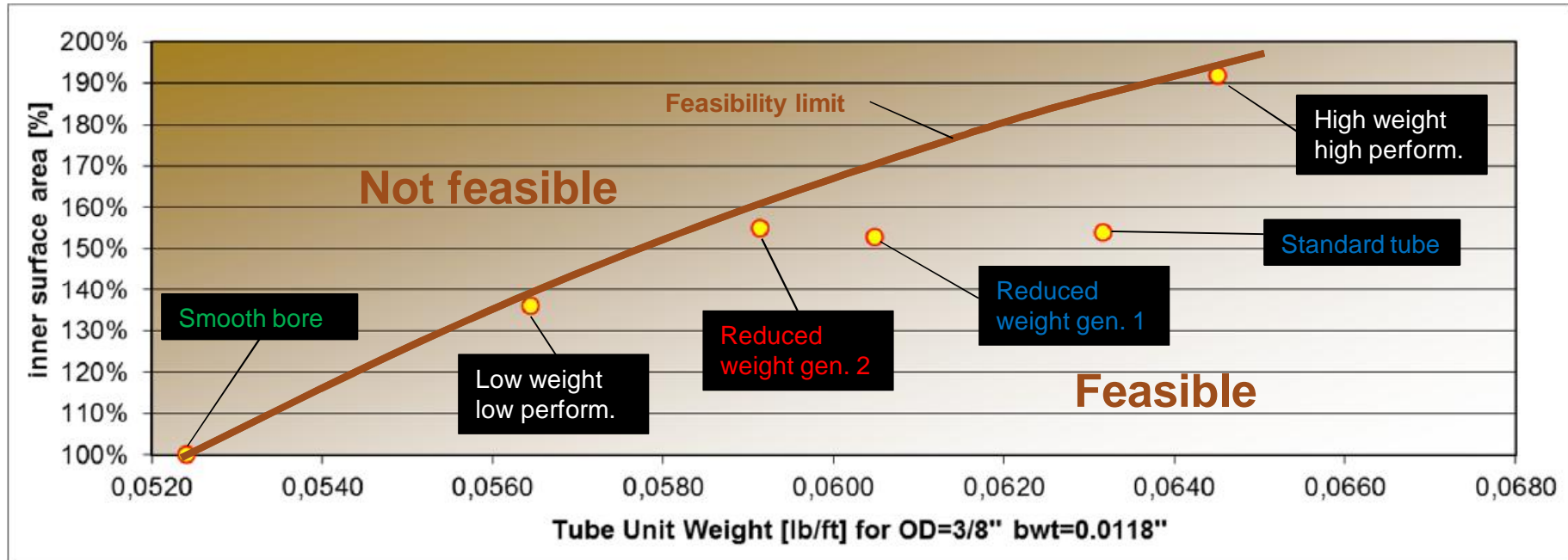


### Results:

- Ø Higher capacity of heat exchanger (HEX)
- Ø Improved energy efficiency
- Ø Reduced size of heat exchanger possible
  - smaller footprint
  - less material costs



# Correlation of tube unit weight and inside surface enhancement of IGT



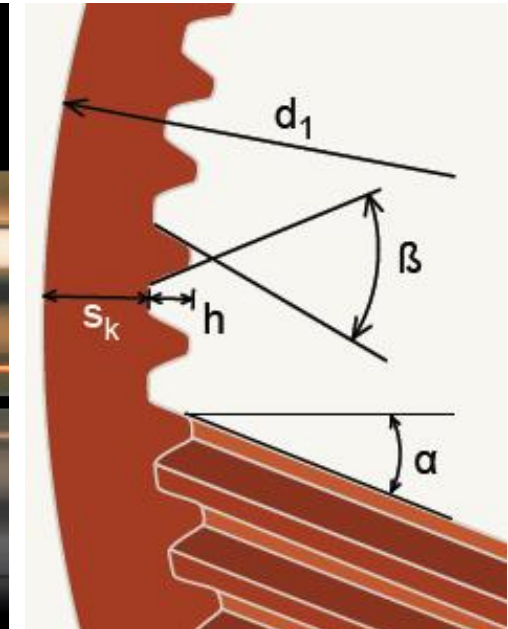
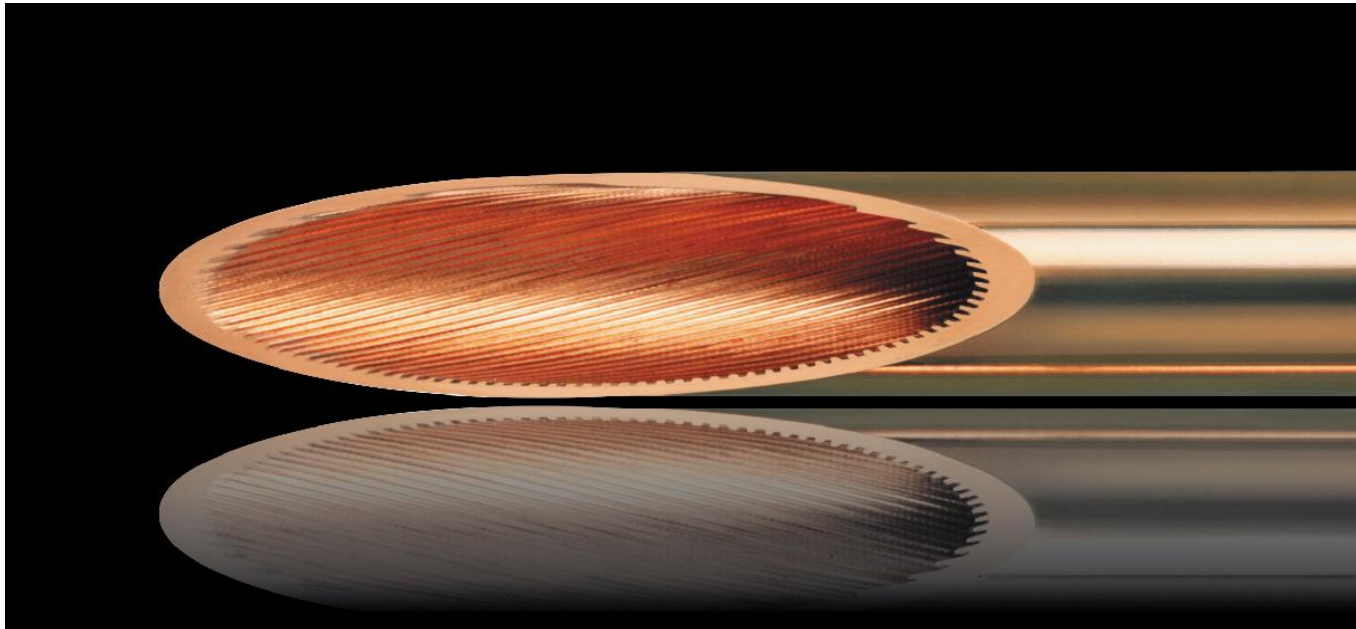
- Ø Lower inside surface results in lower unit weight
- Ø High performing tubes are heavier

**Result:** Tube type should be customized to achieve optimum unit weight at required performance level

slide 6



## Parameters of fin patterns



$h$ ... fin height

$n$ ... number of fins

$\alpha$ ... lead angle (helical angle)

$\beta$ ... apex angle (fin angle)

$d_1$ ... outside diameter

$s_k$ ... bottom wall thickness

slide 7

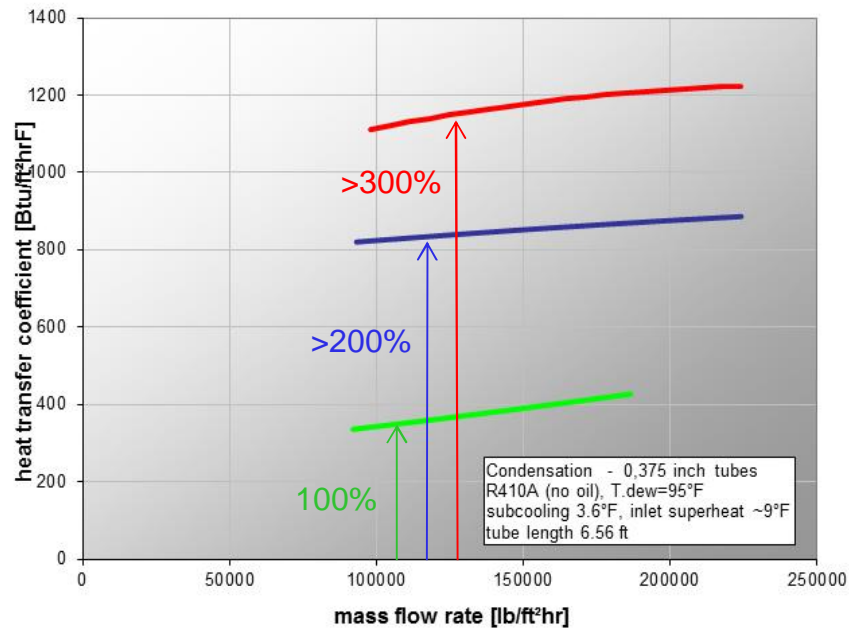


# Optimized tube design for increased tube side performance

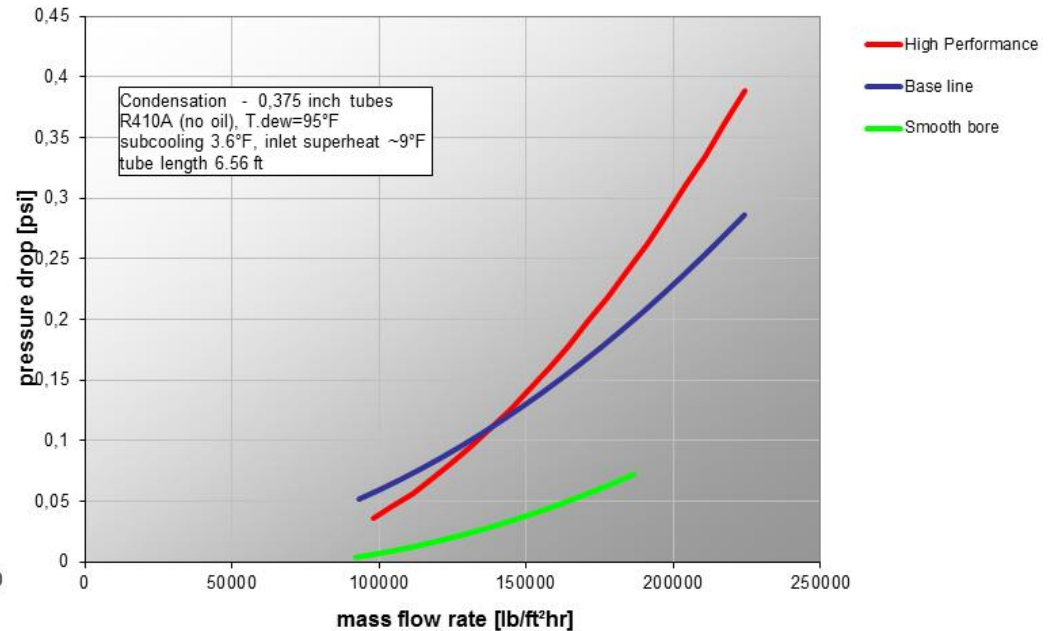
Example: Condensation of R410A in 3/8" diameter tubes

Description	Outside Diam. [inch]	Bottom wall [inch]	Fin height [inch]	Number of fins	Lead angle [deg]	Unit weight increase	Surface increase
Plain tube - Reference	3/8	0,0118	0,0000	0	0	100%	100%
Industry standard - Base line	3/8	0,0118	0,0079	60	18	120%	154%
High performance config.	3/8	0,0118	0,0079	85	30	123%	192%

Heat transfer performance



Total pressure drop

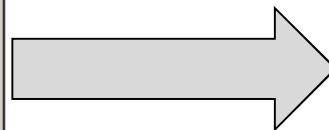
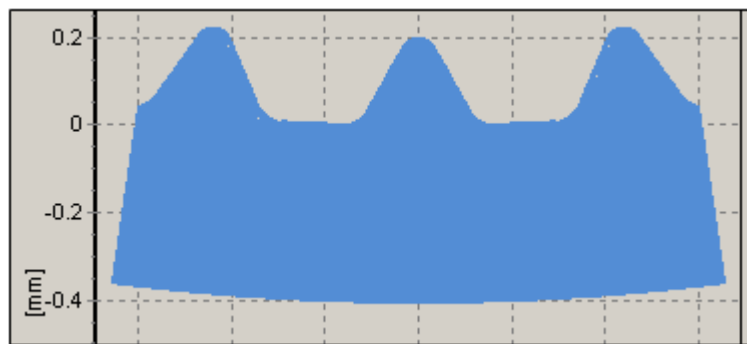




## Optimized tube design for reduced unit weight

Example cuprolite®: Light weight drop-in for 3/8“ diameter IGTs

Description	Outside Diam. [inch]	Bottom wall [inch]	Fin height [inch]	Number of fins	Lead angle [deg]	Apex angle [deg]	Unit weight increase
Plain tube - Reference	3/8	0,0118	0,0000	0	0	0	100%
Industry standard - Base line	3/8	0,0118	0,0079	60	18	53	120%
Light weight - cuprolite	3/8	0,0118	0,0079	50	25	25	113%



- Ø Slimmer fin, decreased number of fins, higher lead angle
- Ø Comparable performance level
- Ø Fin height unchanged → Drop-in

**Result:** Unit weight reduction of > 6%

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## Reduction of bottom wall thickness

Example: 3/8" x 0.0118" reduced to 0.011" bottom wall thickness

Description	Outside diam. [inch]	Bottom wall [inch]	Unit weight factor
Base line	3/8	0,0118	100%
Reduced wall	3/8	0,0110	93%

Issues to be considered when reducing the wall thickness:

- Ø Tools for bending & expansion may have to be replaced
- Ø Pressure level requirements must be checked

**Result: metal cost savings of ~7%**



## Miniaturization: Decreased outside diameter

Example: Optimization of 5.7 ton condenser implementing microgroove technology

Description	Outside diameter	Bottom wall [inch]	Unit weight decrease	Required tube length	Material ratio	Refrigerant charge
Base line	3/8"	0,0118	100%	280 ft	100%	100%
microgroove	5 mm	0,0083	36%	420 ft	54%	44%

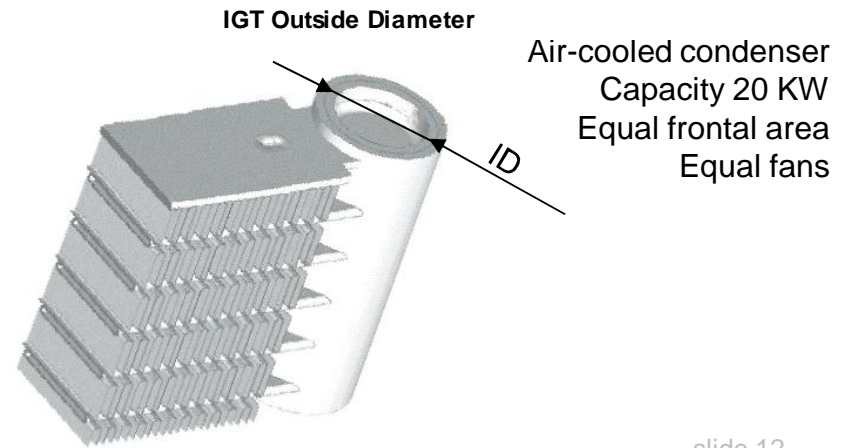
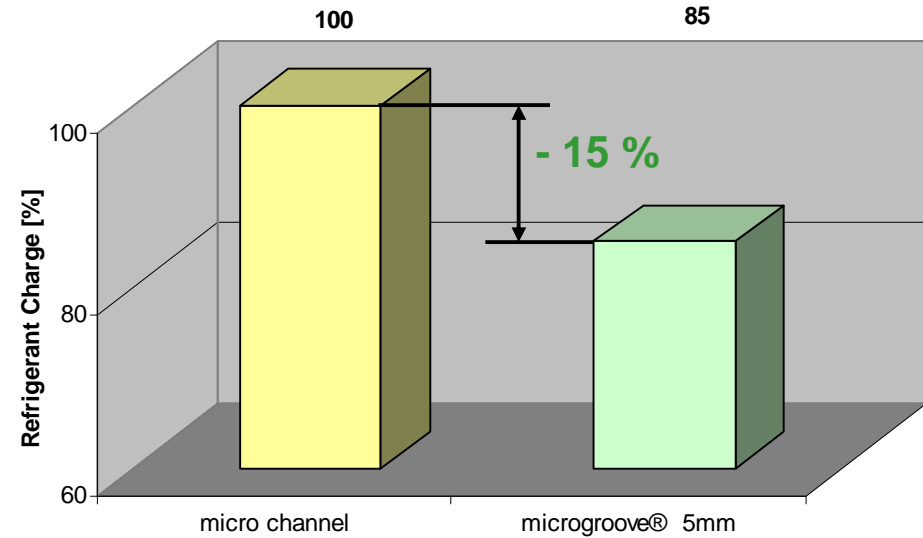
- Ø **Cuts metal costs** nearly into half
- Ø **Improves energy efficiency**
- Ø Significantly **reduces** heat exchanger **weight**
- Ø **Maintains all advantages of copper** tube based heat exchangers
- Ø **Reduces refrigerant charge** by >50%



# microgroove<sup>®</sup> - Refrigerant Charge



Influence of Tube Outside Diameter on Refrigerant Charge



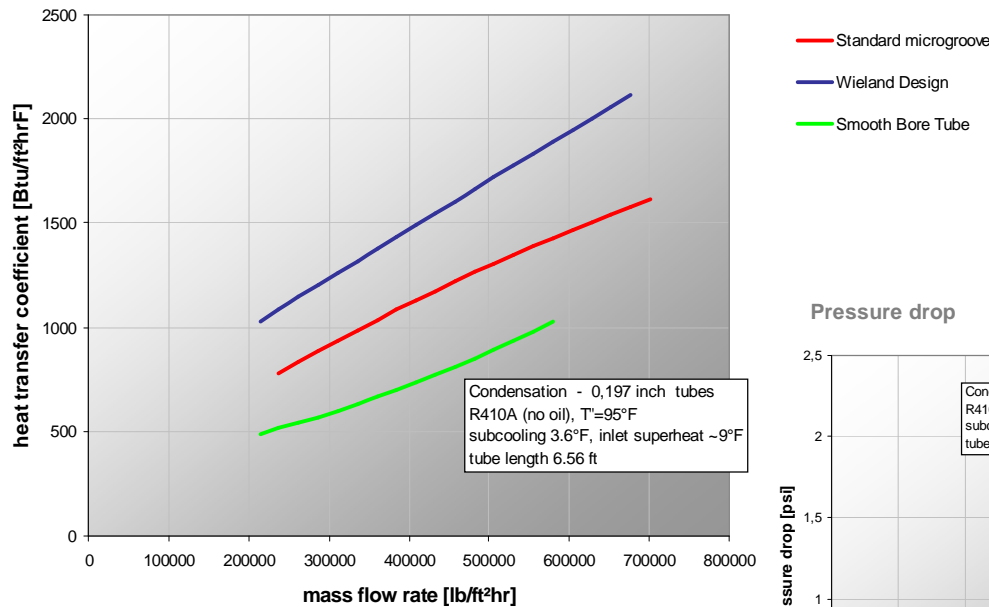
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# Development of IGT on customer demand

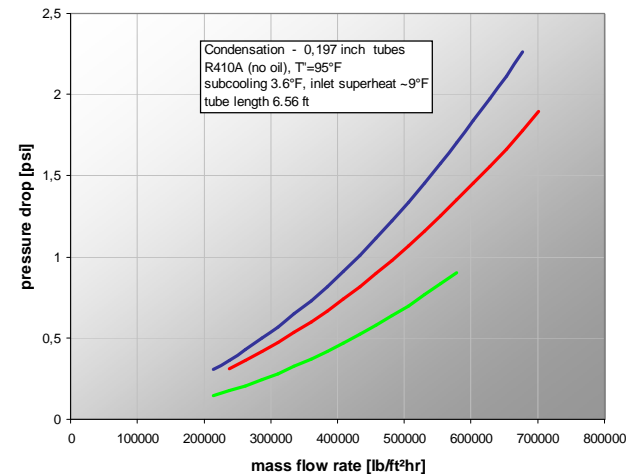
Example: Development task: Design optimized microgroove fin pattern for condensation of R410A

Heat transfer performance



Significant heat transfer enhancement in **condensation** mode was achieved.

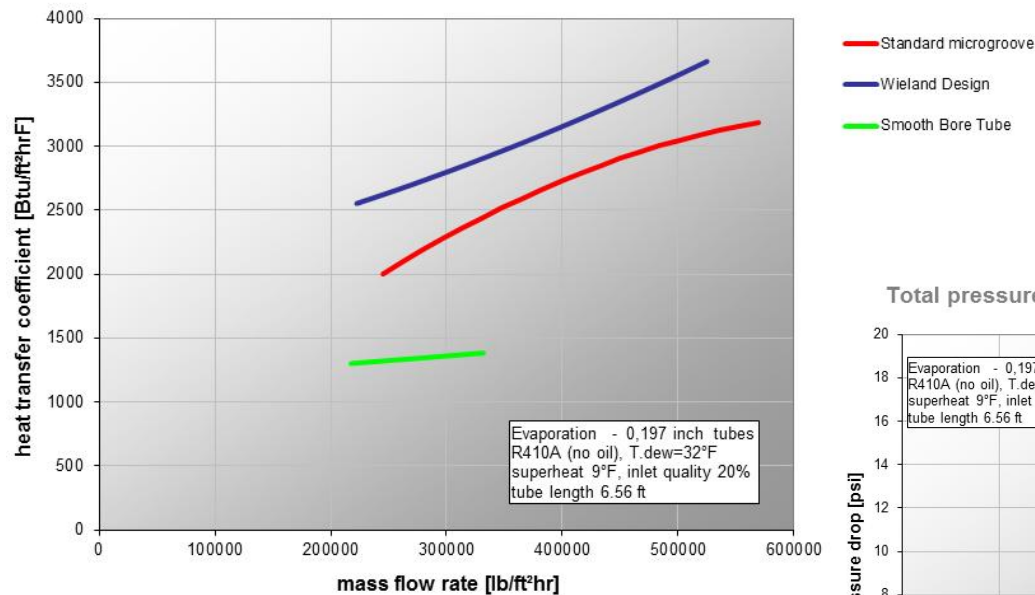
Pressure drop



# Development of IGT on customer demand

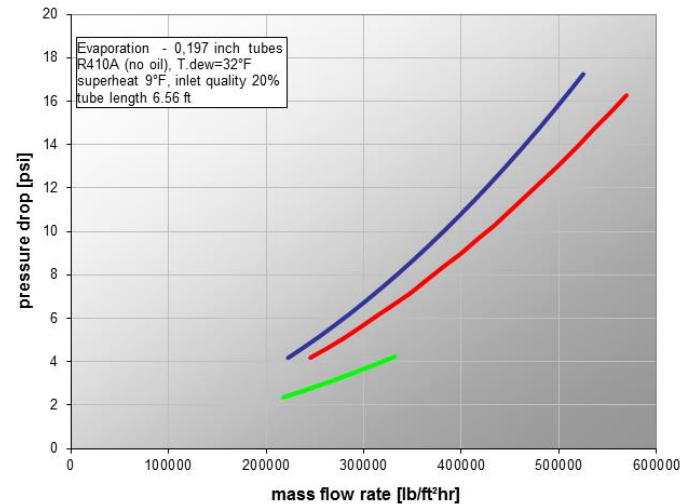
Example: Development task: Design optimized microgroove fin pattern for condensation of R410A

Heat transfer performance



Moderate enhancement for **evaporation** compared to standard microgroove, as this design favors condensation.

Total pressure drop



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Thank you for your attention!

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